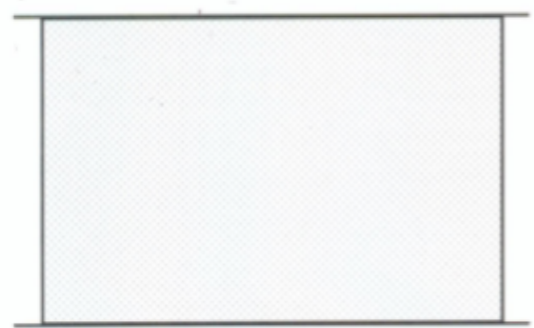
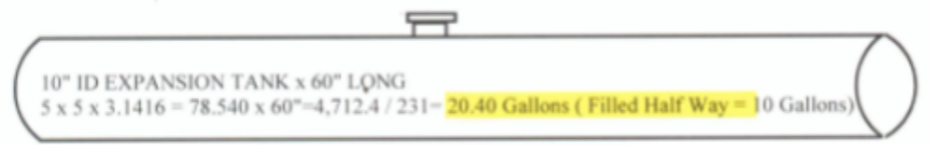
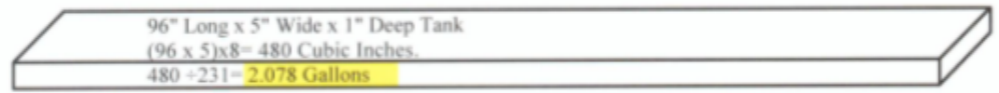
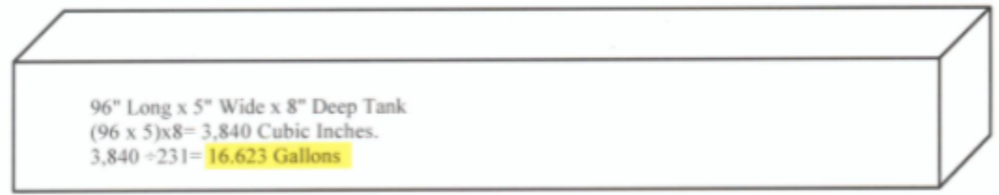
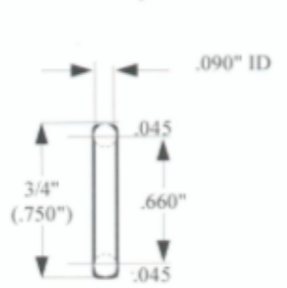


LET'S TIE IT ALL TOGETHER  
AND PUT IT TO USE.





Radiator has 4 cores.  
 Each Core is 48" Tall x 45" Wide x 4-5/8" Deep  
 Has 6 rows of 3/4" Tubes on 5/8" Centers  
 So  $45 \div .625 = 72$  Tubes per Row  
 $72 \text{ Tubes} \times 6 \text{ Rows} = 432 \text{ Tubes}$   
 $432 \text{ Tubes} \times 48" \text{ Long} = 20,736"$  of tubes per core



$.045 \times .045 = .002025$   
 $.002025 \times 3.1416 = .006362$   
 $.090" \times .660" = .0594$   
 $.006362 + .0594 = .065762$  Square Inches  
 $.065762$  Square Inches x 48" Tall Tubes = 3.157 Cubic Inches  
 $3.157$  Cubic Inches x 432 Tubes = 1364 Cubic Inches on ID's of tubes per core.  
 4 Cores x 1364 Cubic Inches = 5,456 Cubic inches total  $\div 231 = 23.619$  Gallons

$16.623 + 2.078 + 16.623 + 10 + 23.619 + 73.439 = 142$  Gallons



BTU



## Basic Engine

### 8V-71

### 8V-71T

Basic engine model	7083-7005				7083-7305			
	Two Cycle				Two Cycle			
Engine type	8				8			
No. of cylinders	4.25 x 5 (108 x 127)				4.25 x 5 (108 x 127)			
Bore & stroke—in (mm)	568 (9.32)				568 (9.32)			
Displacement—cu in (litres)	Standby				Standby			
Application	60 Hz @ 1800		50 Hz @ 1500		60 Hz @ 1800		50 Hz @ 1500	
Frequency @ rpm								
Rated Power, less fan @ 85°F and 500 ft SAE—BHP (kW)*	285	(213)	239	(178)	370	(276)	323	(241)
kW Rating max. @ P.F. 1.0**	200		165		255		220	
Generator efficiency assumed—%	94		94		93		93	
Compression ratio	18.7 to 1		18.7 to 1		17 to 1		17 to 1	
Piston speed—ft/min (m/sec)	1500	(7.62)	1250	(6.35)	1500	(7.62)	1250	(6.35)
No. of main bearings	5		5		5		5	
Approx. net weight dry—lbs (kg)***	5400	(2449)	5400	(2449)	5500	(2495)	5500	(2495)
<b>Air and exhaust system:</b>								
Combustion air requirements—cfm (m <sup>3</sup> /min)	753	(21)	631	(18)	1130	(32)	930	(26)
Max. air intake restriction—in H <sub>2</sub> O (kPa)	25.0	(6.22)	18.0	(4.48)	14.0	(3.48)	10.5	(2.61)
Exhaust gas temp. @ Rated BHP—°F (°C)								
Engine manifold dry	970	(521.1)	955	(512.8)	865	(462.8)	895	(479.4)
Engine manifold wet	920	(493.3)	910	(487.8)				
Exhaust gas flow @ Rated BHP—cfm (m <sup>3</sup> /min)								
Engine manifold dry	1976	(56)	1636	(46)	2747	(78)	2312	(65)
Engine manifold wet	1907	(54)	1586	(45)				
Max. exhaust back press. allowable—in Hg (kPa)	3.3	(11.17)	2.3	(7.79)	2.0	(6.77)	1.4	(4.74)
Exhaust outlet I.D.—in (mm)								
Engine manifold dry	3.5	(88.9)	3.5	(88.9)				
Engine manifold wet	4	(101.60)	4	(101.60)				
Rec. stack single outlet minimum	5	(127)	5	(127)	5	(127)	5	(127)
<b>Cooling system:</b>								
Basic engine water capacity—gal (litres)	8	(30.28)	8	(30.28)	8	(30.28)	8	(30.28)
Jacket water flow—gpm (litres/min)	110	(416.39)	92	(348.26)	110	(416.39)	92	(348.26)
Jacket water temp., normal operation—°F (°C)	170-185	(76.7-85.0)	170-185	(76.7-85.0)	170-185	(76.7-85.0)	170-185	(76.7-85.0)
<b>Heat rejection to jacket water @ Rated BHP—</b>								
Btu/min (W)								
Exhaust manifold dry	8550	(150345)	7170	(126079)	12210	(214704)	10659	(187431)
Exhaust manifold wet	9690	(170391)	8126	(142890)				
Engine heat radiated @ Rated BHP—Btu/min (W)								
Engine manifold dry	2208	(38827)	1910	(33594)	2186	(38440)	1898	(33379)
Engine manifold wet	1877	(33003)	1624	(28555)				
Max. static head @ water pump inlet—ft H <sub>2</sub> O (kPa)	30	(89.58)	30	(89.58)	30	(89.58)	30	(89.58)
Max. heat exchanger raw water press.—psi (kPa)	65	(448.18)	65	(448.18)	65	(448.18)	65	(448.18)
Generator heat radiated to room @ Rated BHP—								
Btu/min (W)†	728	(12801)	708	(12450)	1094	(19237)	1090	(19167)
Air required to radiator—cfm (m <sup>3</sup> /min)††	14500	(411)	11000	(312)	18000	(510)	14500	(411)
Static pressure for air flow—in H <sub>2</sub> O (kPa)	1.0	(.25)	.75	(.19)	1.1	(.27)	1.0	(.25)
<b>Fuel system:</b>								
Fuel pump max. suction, clean system—in Hg (kPa)	6	(20.32)	6	(20.32)	6	(20.32)	6	(20.32)
Fuel quantity pumped—gph (litres/hr)	90	(340.69)	90	(340.69)	90	(340.69)	90	(340.69)
<b>Lubrication-system:</b>								
Oil pan capacity—qts (litres)†††	23	(21.77)	23	(21.77)	23	(21.77)	23	(21.77)
<b>Starting system:</b>								
Electric motors—quantity	1		1		1		1	
Voltage††††	24		24		24		24	
Battery recommended capacity—amp/hr	205 [2]		205 [2]		205 [2]		205 [2]	
Engine rolling current @ 32°F (0.0°C)—amps	845		845		845		845	

\*Nominal basic engine horsepower rating.

\*\*Maximum kW rating at assumed generator efficiency.

\*\*\*Radiator-cooled set.



### Cooling system:

Basic engine water capacity—gal (litres)	8
Jacket water flow—gpm (litres/min)	110
Jacket water temp., normal operation—°F (°C)	170-185
Heat rejection to jacket water @ Rated BHP—	
Btu/min (W)	
Exhaust manifold dry	8550
Exhaust manifold wet	9690
Engine heat radiated @ Rated BHP—Btu/min (W)	
Engine manifold dry	2208
Engine manifold wet	1877
Max. static head @ water pump inlet—ft H <sub>2</sub> O (kPa)	30
Max. heat exchanger raw water press.—psi (kPa)	65
Generator heat radiated to room @ Rated BHP—	
Btu/min (W)†	728
Air required to radiator—cfm (m <sup>3</sup> /min)††	14500
Static pressure for air flow—in H <sub>2</sub> O (kPa)	1.0



285 HP/ 200 kW at 1800 RPM

Jacket water flow 110 GPM

Jacket water temp 170-185F

Heat Rejection 9690 BTU/ MIN



Water weighs 8.34 pounds/ gal  
1 Btu is heat to raise 1 LB water  
1 Degree F.

110 Gallons  $\times$  8.34 = 917.4 LB.

9690 btu/min / 917.4 LB Water  
= 10.56 Degree F.



So at Full Load if the water goes  
Into the radiator at 180F and  
Comes out 10.56F cooler the  
Radiator is working "Good  
Enough".



The mechanic said 14 degree F cooler.

$$917.4 \times 14 = 12,843.6 \text{ BTU's.}$$



Jacket water temp., normal operation		(85.0)
Heat rejection to jacket water @ Rated BHP—		
Btu/min (W)		
Exhaust manifold dry	8550	(150345)
Exhaust manifold wet	9690	(170391)
Engine heat radiated @ Rated BHP—		
Btu/min (W)		
Engine manifold dry	2208	(38827)
Engine manifold wet	1877	(33003)
Max. static head @ water pump—	30	(89.58)
in H <sub>2</sub> O (kPa)		
Max. heat exchanger raw water pressure—	65	(448.18)
psi (kPa)		
Generator heat radiated to room @ Rated BHP—		
Btu/min (W)†	728	(12801)
Air required to radiator—cfm (m <sup>3</sup> /min)††	14500	(411)
Static pressure for air flow—in H <sub>2</sub> O (kPa)	1.0	(.25)



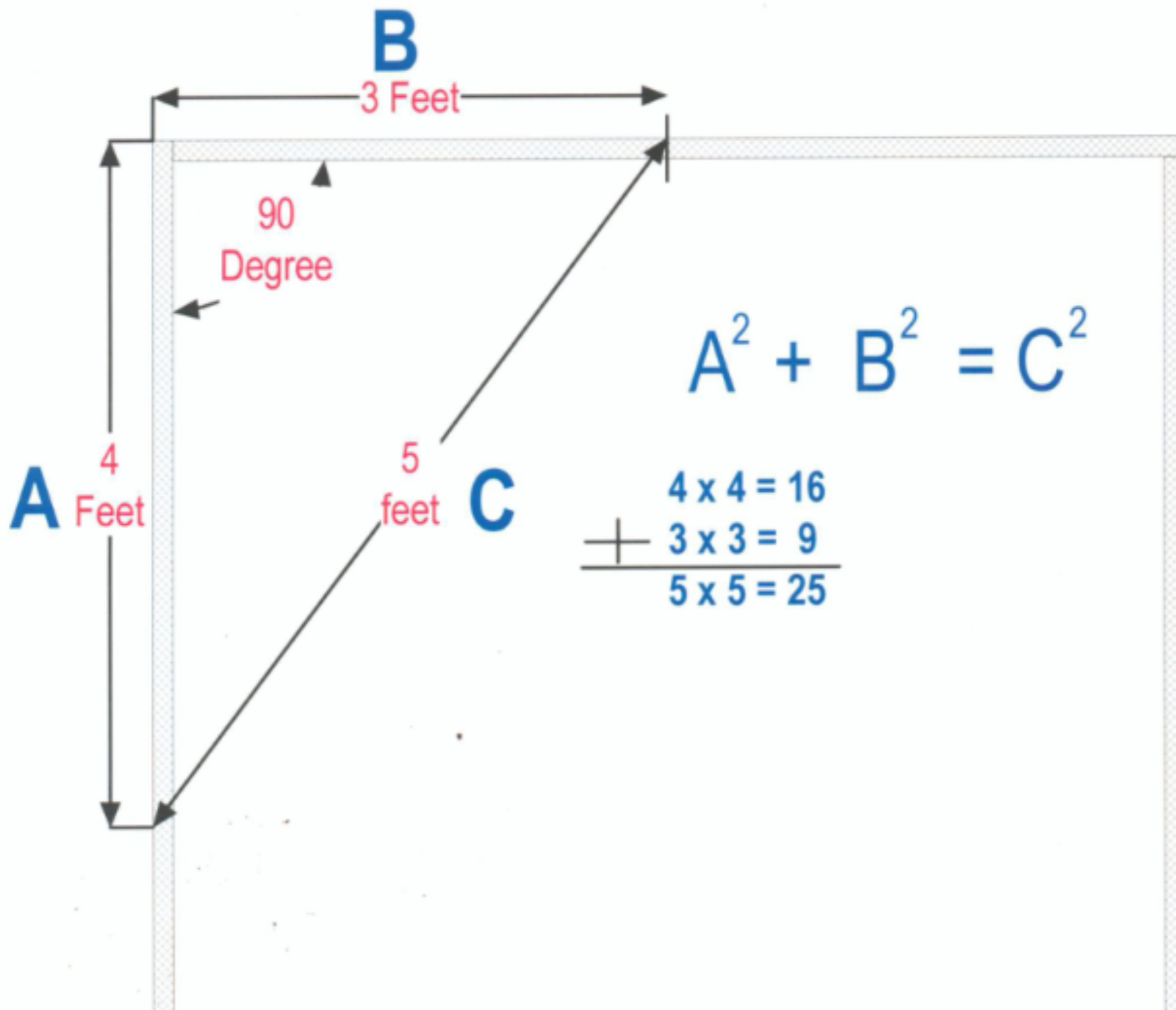
Spec sheet says 9690 with  
A water cooled exhaust manifold!

Not the radiators problem. Perhaps something  
mechanical.



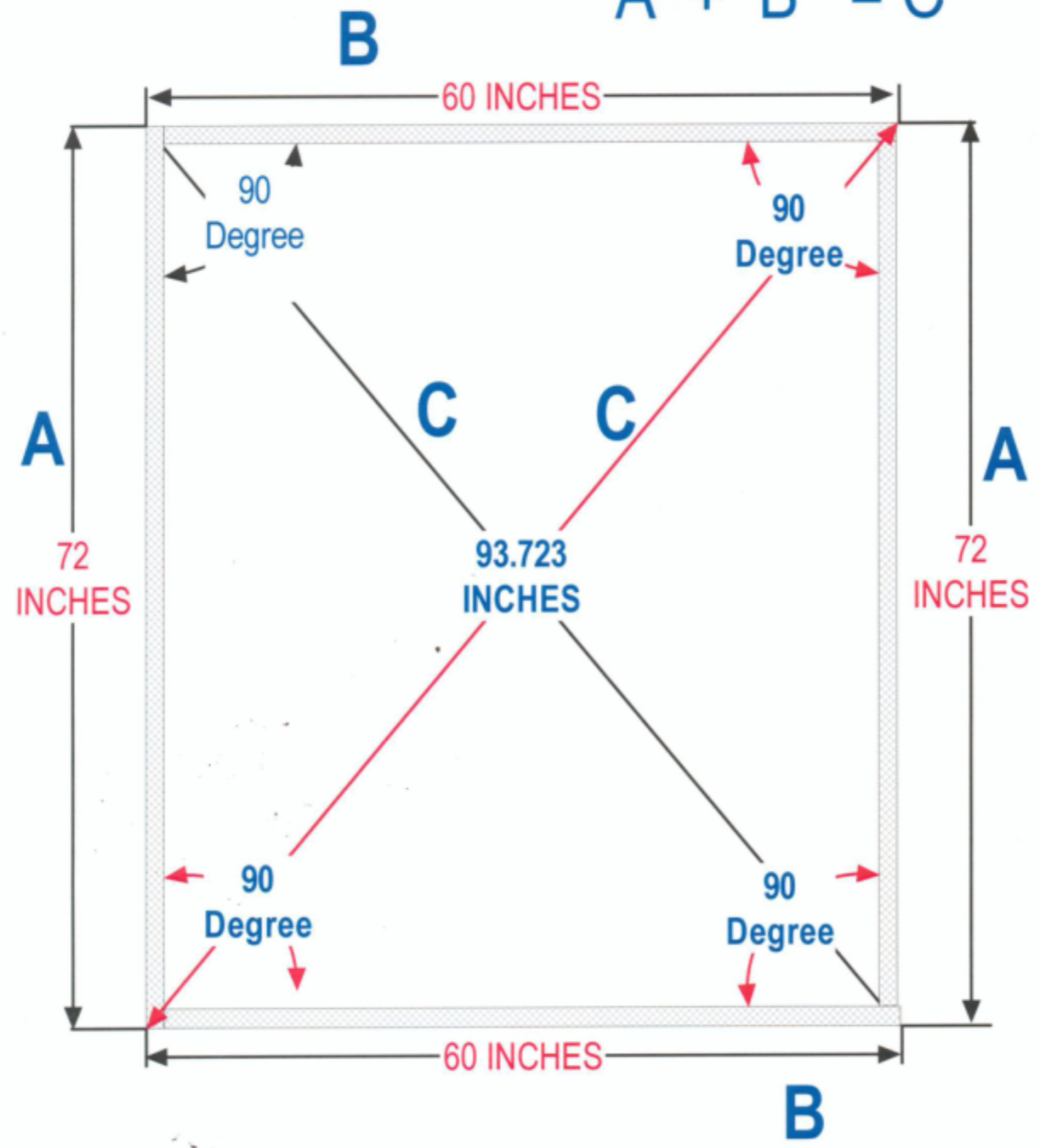
# GEOOMETRY







$$A^2 + B^2 = C^2$$

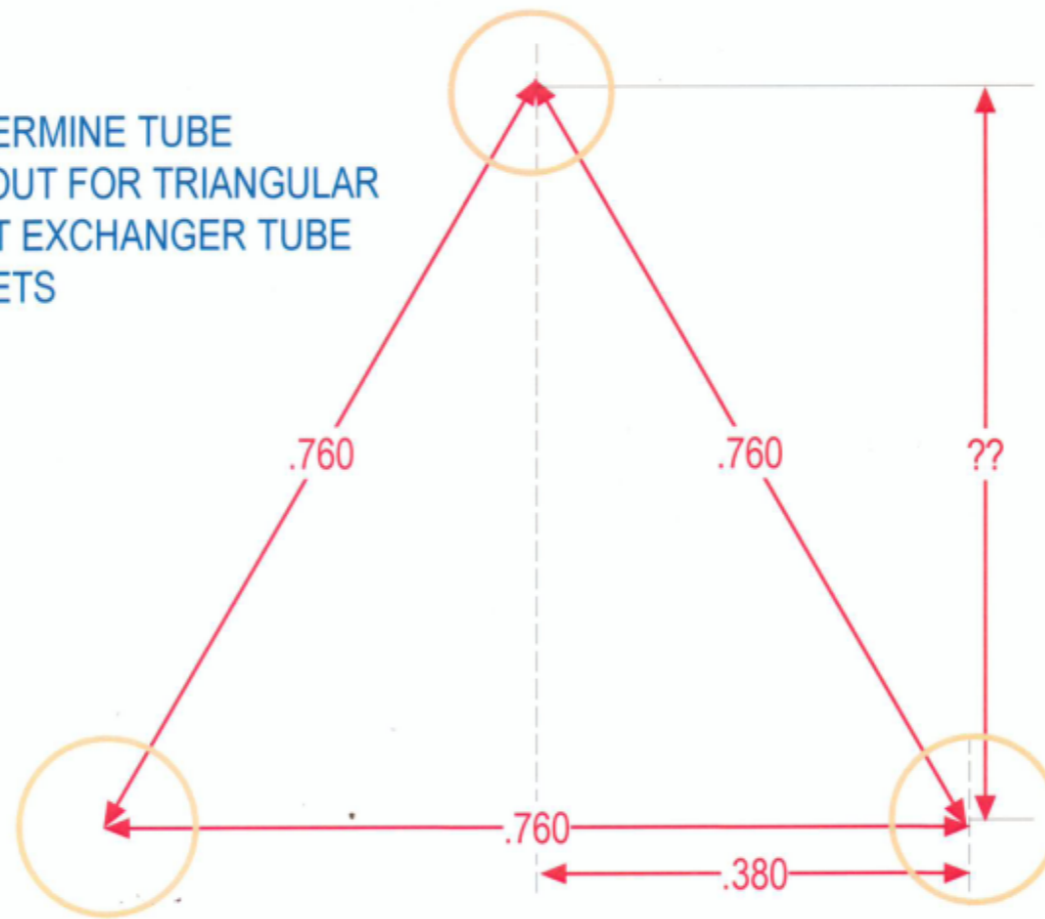








DETERMINE TUBE  
LAYOUT FOR TRIANGULAR  
HEAT EXCHANGER TUBE  
SHEETS



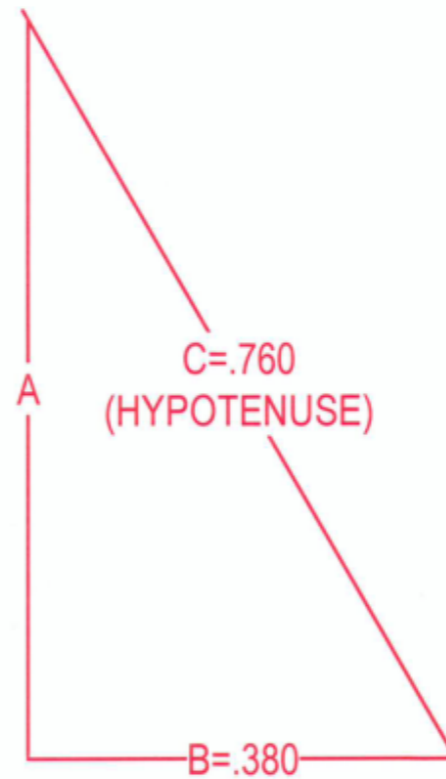
$$C = .760 \times .760 = .5776$$

$$- B = .380 \times .380 = .1444$$

$$A = \sqrt{X} \text{ of } .4332$$

$$\sqrt{\quad} \text{ of } .4332 = .658$$

$$\text{So } A = .658''$$







DEWALT

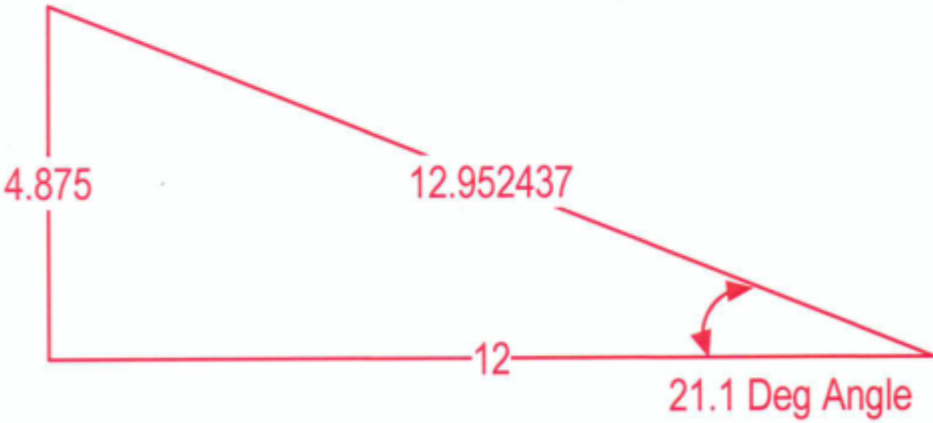
MAXIMUM  
Folgers  
CLASSIC







DETERMINE FAN BLADE ANGLE



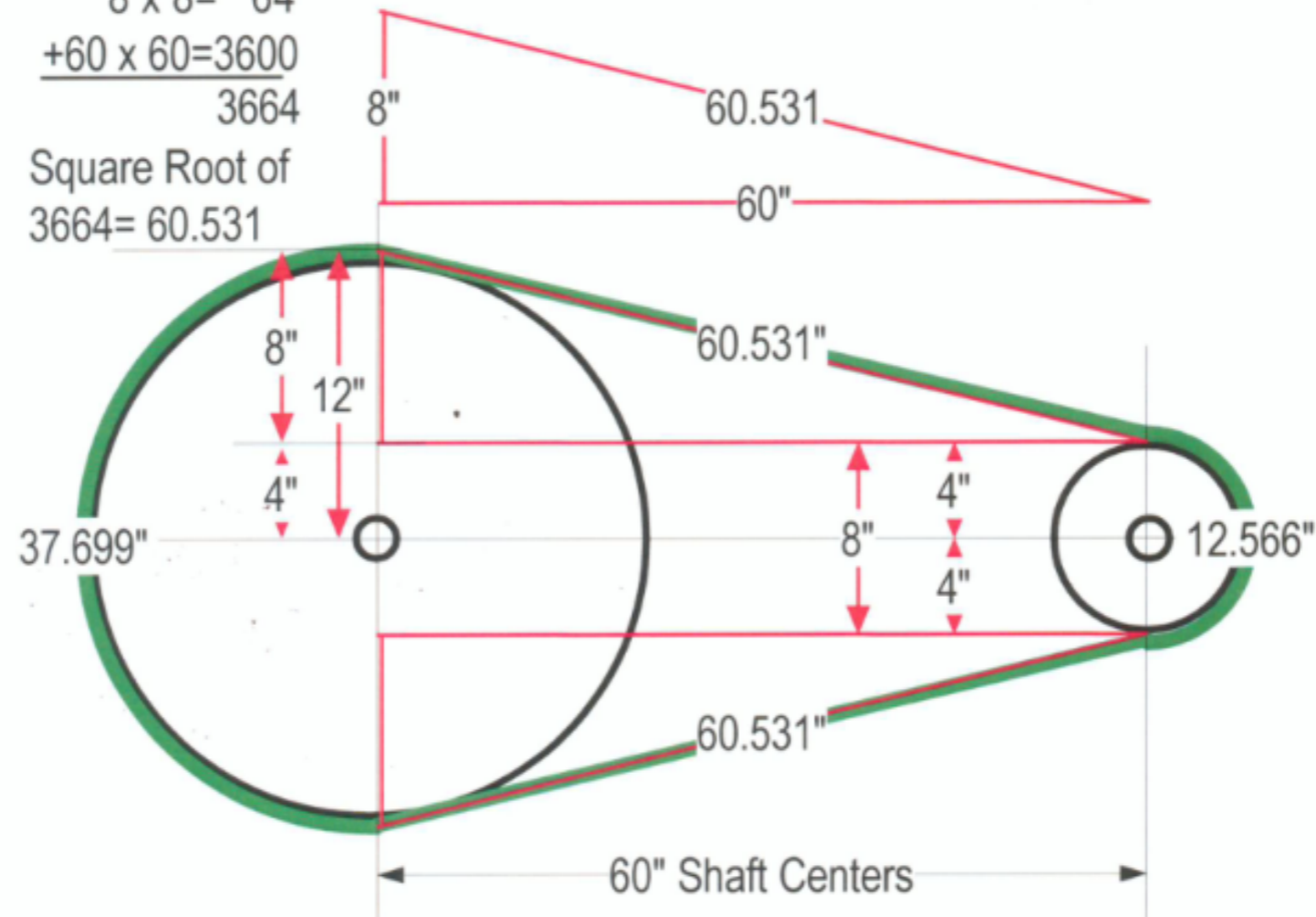


# BELT LENGTH

$$A^2 + B^2 = C^2$$

$$\begin{array}{r} 8 \times 8 = 64 \\ +60 \times 60 = 3600 \\ \hline 3664 \end{array}$$

$$\text{Square Root of } 3664 = 60.531$$



24" Diameter Pulley  
 Circuference = Pi x Diameter  
 $3.1416 \times 24 = 75.398"$   
 Half of Circumference = 37.699"

8" Diameter Pulley  
 Circuference = Pi x Diameter  
 $3.1416 \times 8 = 25.132"$   
 Half of Circumference = 12.566"

$$\text{Belt Length} = 37.699" + 60.531" + 60.531" + 12.566" = 171.327" \text{ (5VX1710 Belt)}$$



5VX171 BELT- YOU KNOW PULLEY SIZES AND BELT SIZE.  
 WHAT DOES SHAFT CENTERS HAVE TO BE?

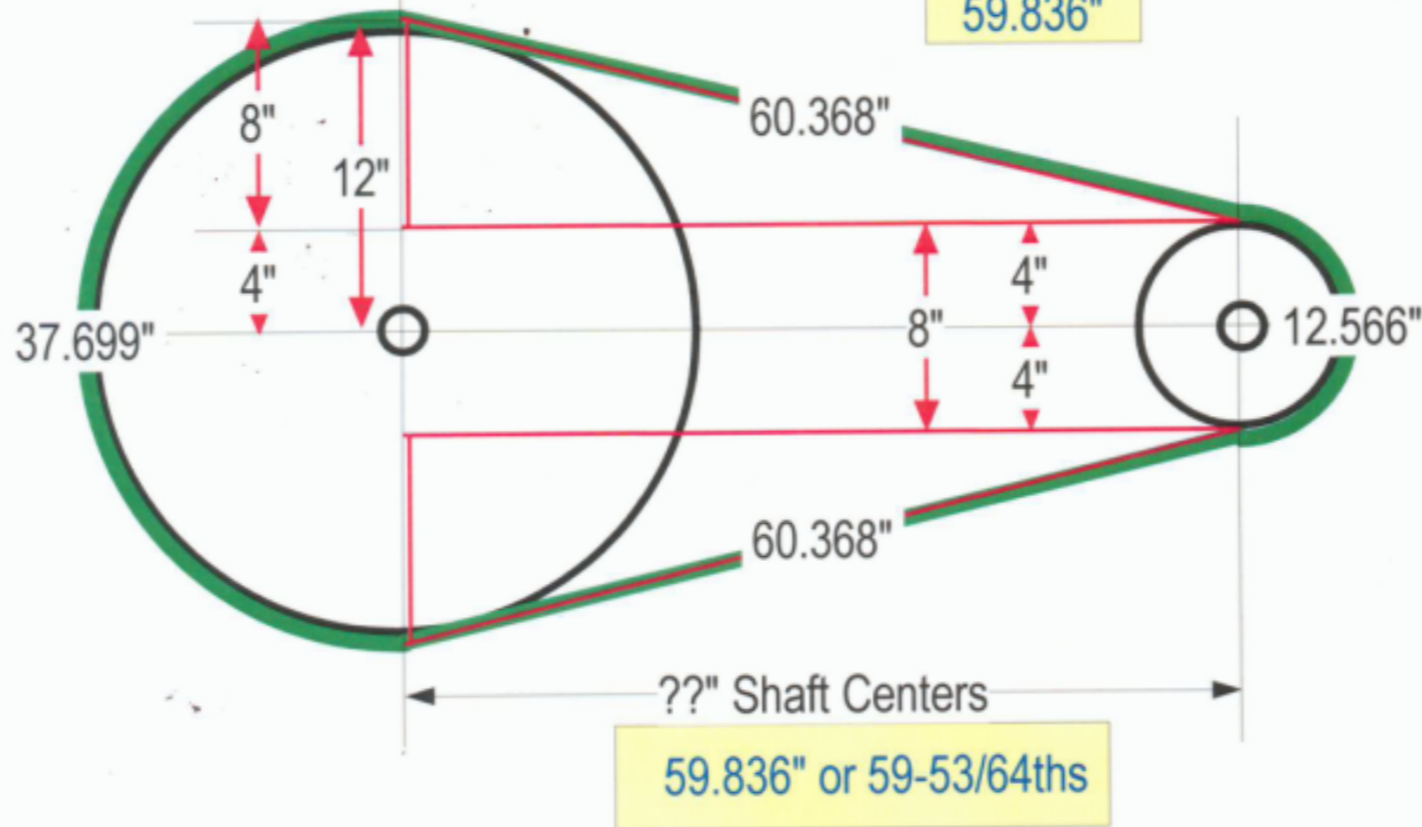
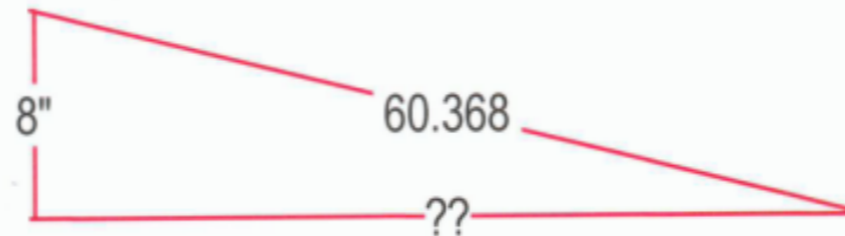
$$171" - (37.699 + 12.566") = 120.735"$$

$$\text{HALF OF } 120.735 = 60.368"$$

# SHAFT CENTERS

$$\frac{60.368 \times 60.368 = 3644.295}{8 \times 8 = 64.000} = 3580.295$$

Square Root of  
 3580.298 = 59.836



24" Diameter Pulley  
 Circuference = Pi x Diameter  
 $3.1416 \times 24 = 75.398"$   
 Half of Circumference = 37.699"

8" Diameter Pulley  
 Circuference = Pi x Diameter  
 $3.1416 \times 8 = 25.132"$   
 Half of Circumference = 12.566"

59.836" or 59-53/64ths





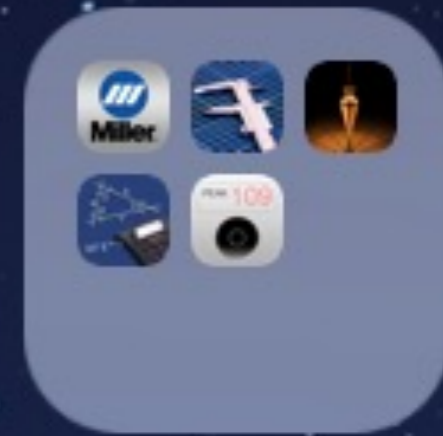
Light



Speedometer



Find iPhone



WORK APPS



STORES



RECOVERY



CATHOLIC



Voice Memos



FaceTime



Draw Free



LinkedIn



Wig It!





WeldSettings



PreSize



Carpenter



TriangleCalc



db meter



right-angled triangle

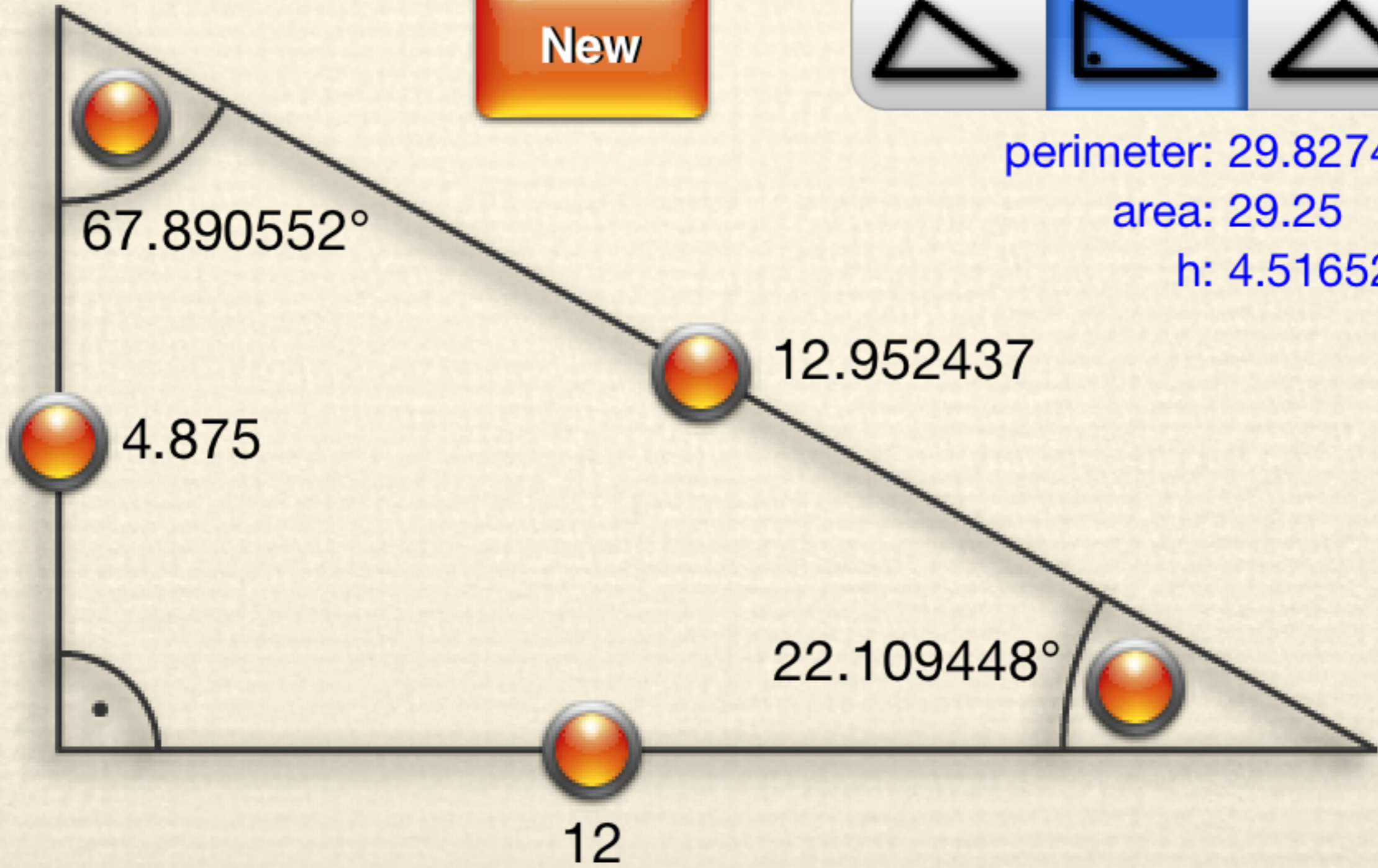
New



perimeter: 29.827437

area: 29.25

h: 4.5165246





22.2°



Hold





# Fan Blade Angle Cheat Sheet

(Fill in          only!

\ D=Ang

A

B

C

D

A

B

2

4

26.56505

1.25

3.625

19.02561

1.875

3.5

28.17859

3

6.5

24.77514

2

9

12.52881

2.25

3.25

34.69515

2.5

4.625

28.39302

2.5

3.1875

38.10758

4

9.5

22.83365

2.25

5.25

23.19859

4.25

8

27.97947

6.875

3.75

61.38954

#DIV/0!

#DIV/0!

#DIV/0!



# FAN TIP SPEED

# CALCULATION CHART

Fill in colored area information only.

7	Blade	Fan	Crank	Engine	Fan Dia./	Feet Per Second		Fan
	8 Diameter	Pully OD	Pully OD	RPM	Engine	Actual	Allowable	RPM
9	33	18	11.025	1200	D353 Cat	105.8327	377	735
10	34	7	10.5	1200	26"6-71	267.036	377	1800
11	40	9	8.5	2100	40"8V92T	346.1578	377	1983.333
12	40	6	6	1800	Sample	314.16	377	1800
13	16	5	6.25	1800	Tetra-Onan	157.08	377	2250
14	56	14.5	9	2250	BJ FRAC	341.2428	342	1396.552
15	26	7.25	9	1500	LMSales	211.2455	377	1862.069
16	34	8.5	7	2100	D Tech	256.564		1729.412
17	30	8	7.375	2100	BJ DDC50	253.4142		1935.938
18	27	7.25	7.5	2100	Sup 6V71	255.9321	377	2172.414
19	40	12	7.5	2300	Sup New Su	250.8917		1437.5
20	40	10.5	8.5	2300		324.9644		1861.905
21	48	16.75	11	1200	SS266A	165.0512		788.0597
22	48	1	0.85	1800		320.4432		1530
23	35.75	1	1	2100	SUPREME	327.5773	377	2100
24	48	15.25	2.875	1725	Shop fan	68.11092		325.2049
25	44	9.5	7.5	2100	HWC Cat	318.2937	377	1657.895
26	40	10	8	1800		251.328		1440
27	40	10	8	2100	Sup 16V71	350.856		1718.182

14-16 FAN

Unocal

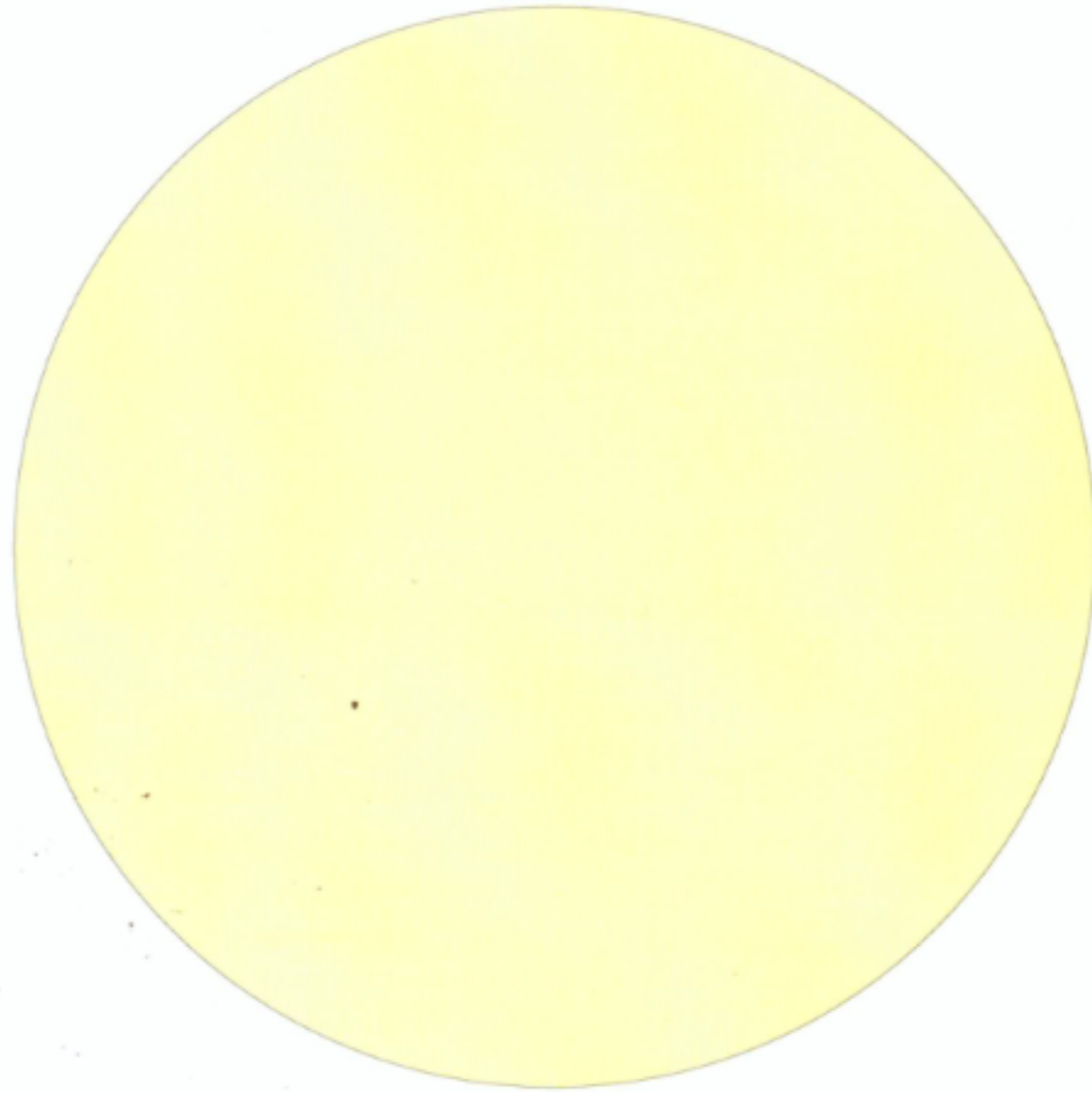
CUMMINS





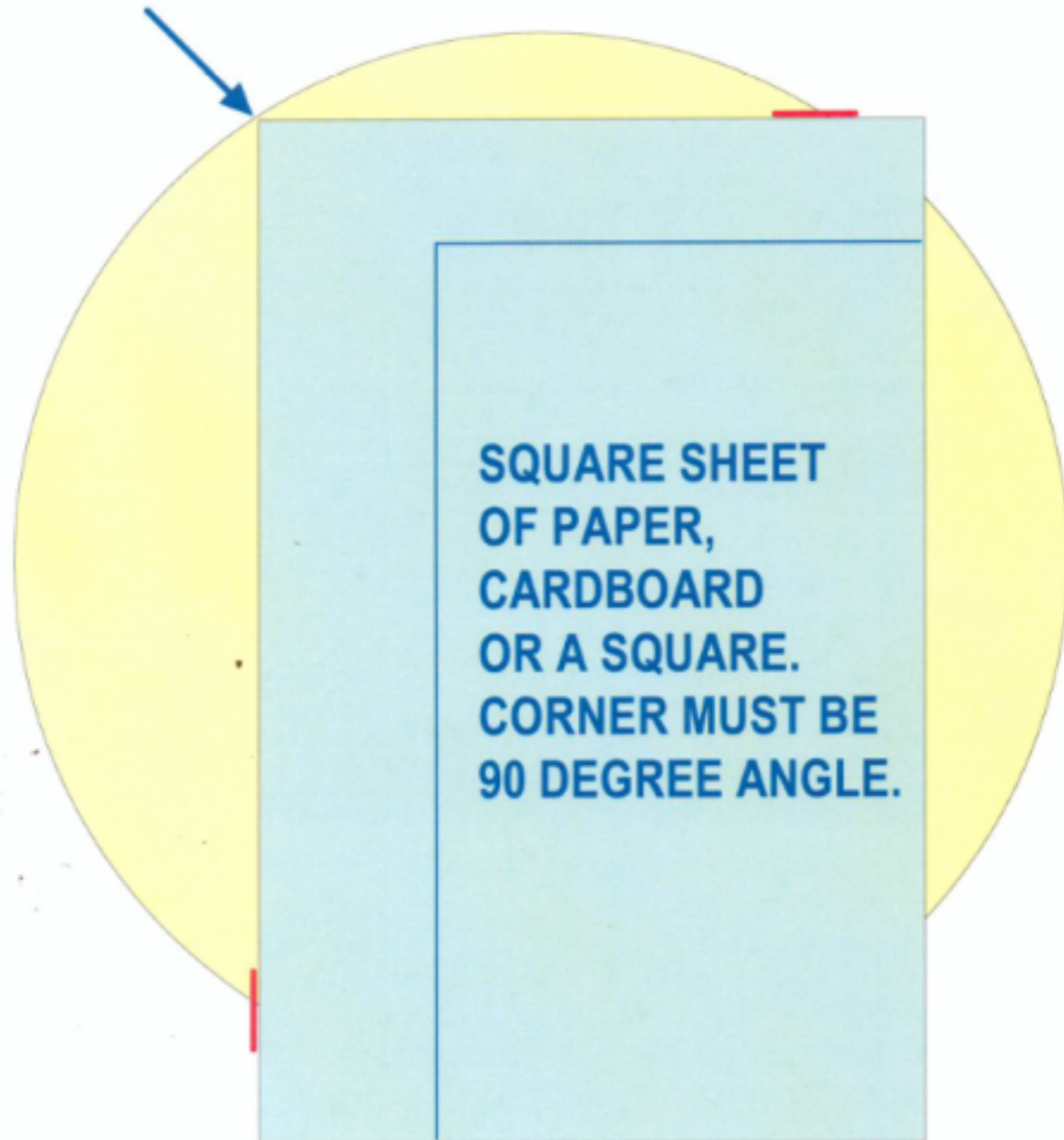


## HOW TO LOCATE THE CENTER POINT OF A CIRCLE



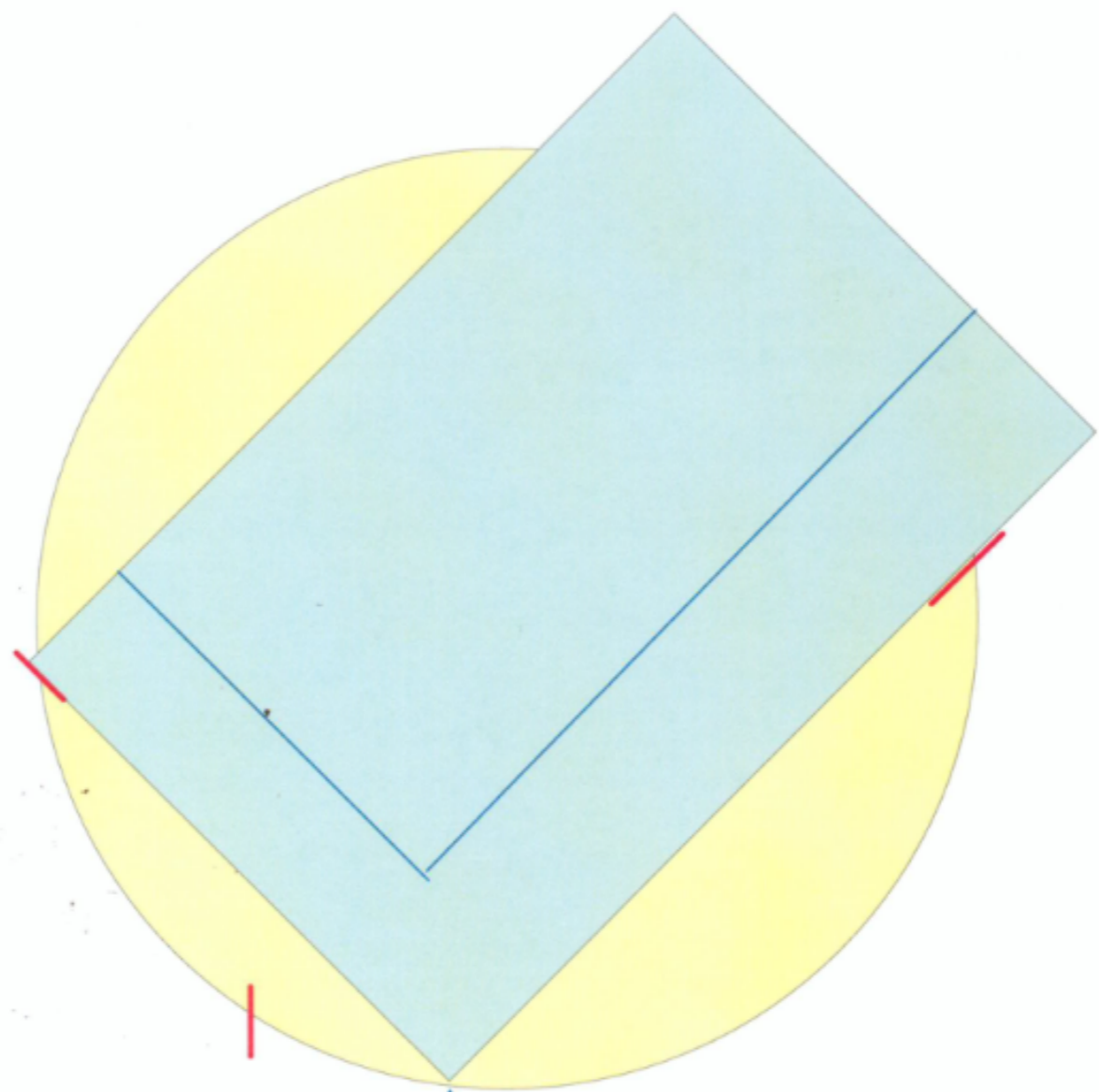


**FLUSH WITH EDGE**



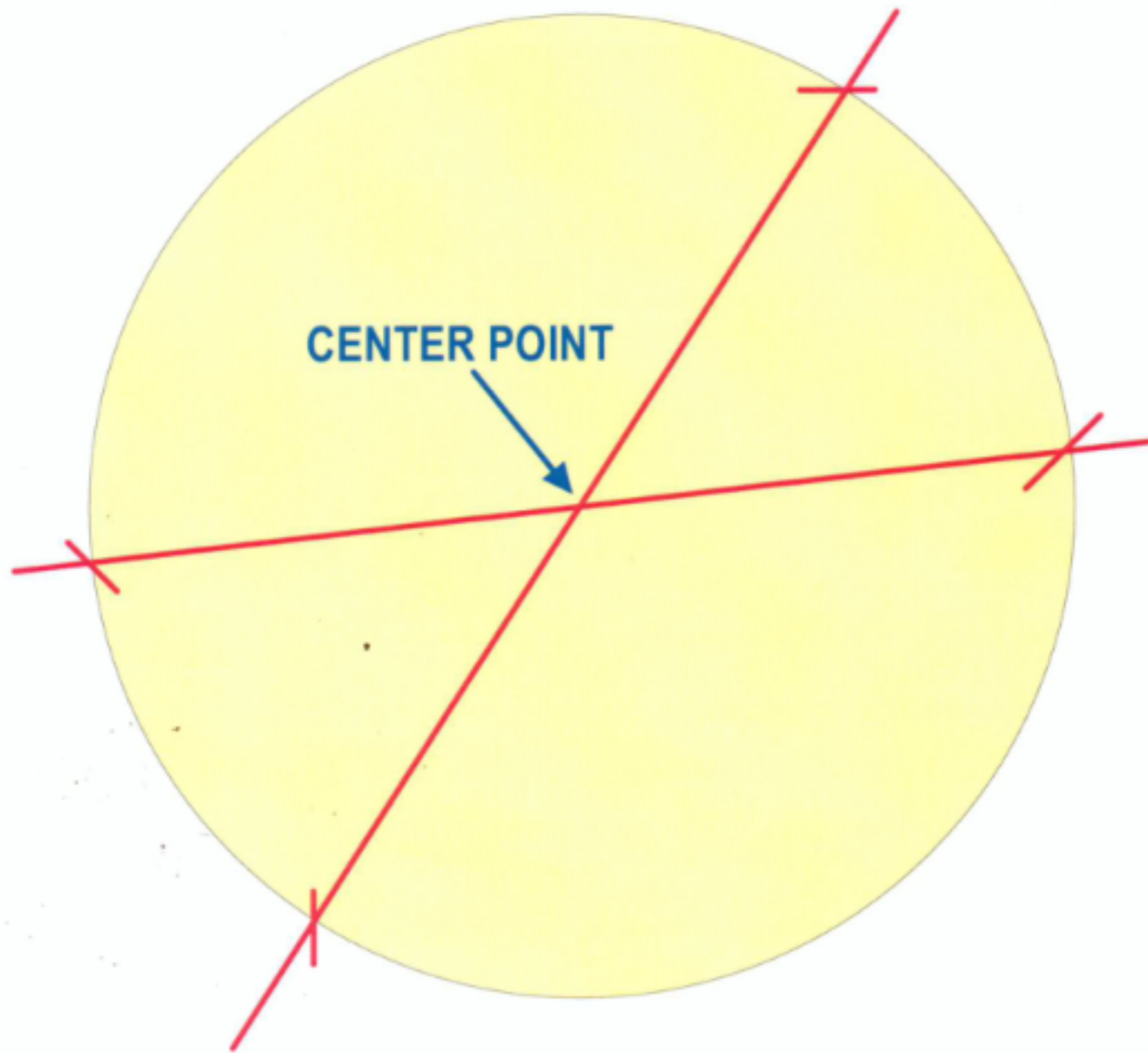
**SQUARE SHEET  
OF PAPER,  
CARDBOARD  
OR A SQUARE.  
CORNER MUST BE  
90 DEGREE ANGLE.**





FLUSH WITH EDGE





CENTER POINT



CUBIC INCH/61 = Cubic Litres

CUBIC LITRES x 61 = Cubic Inch

350 CID/61 = 5.7L

5.7L x 61 = 350 CID



# METRIC TO US CONVERSION WORK SHEET

FROM	UNIT	TO	UNIT	FORMULA	ANSWER	
1680	Kg	TO	Lb.	$1\text{Kg}=2.204623$	3703.77	Lb.
1085	Lb	TO	Kg.	$2.204623\text{Lb}=1\text{Kg}$	492.15	Kg.
103.4	kPa	TO	PSI	$\text{kPa}/100 \times 14.504$	15.00	PSI
60	MM	TO	INCH	$\text{MM}/25.4$	2.362	INCH
50	MM	TO	CM	$\text{MM}/10.00$	5.000	CM
66	INCH	TO	CM	$\text{INCH}/0.39370$	167.640	CM
137.2	CM	TO	INCH	$\text{CM} \times .39370$	54.000	INCH
66	INCH	TO	MM	$\text{INCH} \times 25.4$	1676.40	MM
1	GAL	TO	LIT	$1\text{ GAL}=3.78\text{L}$	3.78	LIT
20000	LIT	TO	GAL	$3.78\text{L}=1\text{GAL}$	5291.01	GAL
200	ML	TO	OZ	$1\text{ ML}=0.03381\text{ OZ}$	6.76	OZ
7	OZ	TO	ML	$0.03381\text{ OZ}=1\text{ML}$	207.04	ML
180	Deg C	TO	Deg F	$(9 \times \text{C}) / 5 + 32$	356	Deg F
122	Deg F	TO	Deg C	$5/9(\text{F}-32)$	50.00	Deg C
138	Knots	TO	MPH	$\text{Knot}/.868$	159.0	MPH
159	MPH	TO	Knots	$\text{MPH} \times .868$	138.01	Knots
9.5	KGM	TO	ft/lb	$\text{KGM} \times 7.236$	68.7	ft/lb
9.5	KGM	TO	in/lb	$\text{KGM} \times 86.81$	824.7	in/lb



# DRILL FEEDS AND SPEEDS

DRILL SIZE		FEED	REV/	INCH/	SUR FT/		SET UP MILL	
FRC	DEC	I.P.R.	MINUTE	MINUTE	MINUTE		RPM	FEED
3/8	0.375	0.007	1018.7	7.1307	100	STEEL	1019	7.1
27/64	0.422	0.008	724.17	5.7934	80	STEEL	724	5.8
17/32	0.531	0.007	575.52	4.0286	80	STEEL	576	4.0
7/8	0.875	0.004	349.26	1.397	80	STEEL	349	1.4
1/4	0.25	0.004	1222.4	4.8896	80	STEEL	1222	4.9
1/2	0.5	0.007	611.2	4.2784	80	STEEL	611	4.3
3/4	0.75	0.007	200	1.4	75	STEEL	200	1.4
1/2	0.5	0.01	382	3.82	50	SPRING	382	3.8
			#DIV/0!	#DIV/0!			#####	#####
			#DIV/0!	#DIV/0!			#####	#####
			#DIV/0!	#DIV/0!			#####	#####
1/4	0.25	0.004	2292	9.168	150	BRASS	2292	9.2
3/8	0.375	0.006	2037.3	12.224	200	BRASS	2037	12.2
5/8	0.625	0.008	1222.4	9.7792	200	BRASS	1222	9.8
1/2	0.5	0.007	1528	10.696	200	BRASS	1528	10.7

GOOD



$$1 \text{ BAR} = 14.504 \text{ PSI}$$

$$1 \text{ BAR} = 100 \text{ kPa}$$

$$\text{CONVERT kPa TO PSI? } \text{Kpa}/100 \times 14.504 = \text{PSI}$$

$$\text{KW to BTU/Hour (Develops) } \underline{\text{KW}} \times 3412.874$$

$$\text{KW to BTU/Min } \underline{\text{KW}} \times 56.8$$

$$1 \text{ HP} = 2545 \text{ BTU/hr}$$

$$\text{Perfect vacuum} = 29.92'' \text{ Mercury}$$

$$\text{Liter/min} \times .26417 = \text{GPM}$$

$$\text{MM to inches? } \text{Inches} / 25.4 = \text{MM}$$

$$\text{Inches to MM? } \text{MM} \times 25.4 = \text{inches}$$

$$\text{Cooling water: } ^\circ\text{F} \times \text{GPM} \times ^\circ\text{F Liquid Temp Drop} \times 8.33 = \text{Btu/Hr.}$$

$$1 \text{ gallon of water weigh } 8.34 \text{ lb}$$

$$1 \text{ pound of fresh water} \times 1.6-1.9 = \text{Pound of seawater}$$

$$\text{Gallon} \times 3.78 = \text{Liter}$$

$$\text{Liter}/3.78 = \text{Gallons}$$

$$\text{Celsius} = \frac{5(\text{F}-32)}{9}$$

$$\text{Fahrenheit} = \frac{9\text{C}}{5} + 32$$

$$\text{Torque } \underline{\text{KGM}} \times 7.236 = \text{ft/lb or } \times 86.81 (7.236 \times 12) = \text{in/lb}$$

$$1 \text{ foot of head} = .433 \text{ PSI}$$



**Pressure Head** Measurement of water pressure based on the water depth. Measurement is stated as "feet of head" or "meters of head". One foot of head is the pressure at the bottom of a 1 foot high column of water, which is also equal to 0.433 PSI. So it's really a measure of the weight of water of a given depth. It doesn't matter how much water is present, the pressure head is only determined by the depth of the water. The water pressure at the bottom of a 2" diameter, 20 foot tall water filled pipe is the same as the water pressure at the bottom of a 20 foot deep lake. (No joke, it's true!)

|



# HEAT EXCHANGER SURFACE AREA

## CALCULATION CHART

Fill in colored area information only.

Tube Diameter	Cooler Type	Tube Count	Discription MFD By	Tube Length	Surface Area	
					Sq. Inch	Sq. Foot
0.375	KTA1150	150	Low HP	17.5	3092.513	21.47578
					0	0
0.25	KTA1150	366	Hi HP	17.5	5030.487	34.93394
0.375	D399	222	CAT	50.5	13207.68	91.71999
0.394	ELI	76	GERMAN	27	2539.946	17.63851
0.375	CONVERT	79.67631	3/8TH	27	2534.4	17.6
0.375		80		27	2544.696	17.6715
1.66	keel cool	96.51826		20	10066.96	69.90946
					0	0
0.375		76.05466		18	1612.8	11.2
0.375		76		15	1343.034	9.326625
0.375	WMI U T	371		84	36714.31	254.9605
0.425		371		84	41609.55	288.9552
2.375		1		482.4896	3600	25
2.375		1		6	44.7678	0.310888
					0	0
0.31	CUM	109		25.75	2733.483	18.98252
0.25		136		25.75	2750.471	19.10049
					0	0
0.625	BWC	118		81	18767.13	130.3273
0.375		220		72	18661.1	129.591
0.375		122		36	5174.215	35.93205

















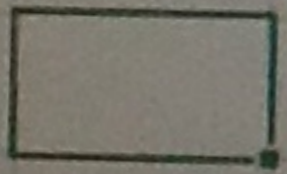


L M N O P Q R S T U W

# CIRCLES

Fill in colored area information only.

DIA. IN INCHES	CIR. IN INCHES	CIR IN FEET							
90	282.743	23.562	23	276	6 3/4	23	FT	6.743	IN
	0.000	0.000		0	0	0	FT	0.000	IN
	0.000	0.000		0	0	0	FT	0.000	IN
	0.000	0.000		0	0	0	FT	0.000	IN
	0.000	0.000		0	0	0	FT	0.000	IN
	0.000	0.000		0	0	0	FT	0.000	IN
	0.000	0.000		0	0	0	FT	0.000	IN
	0.000	0.000		0	0	0	FT	0.000	IN
	0.000	0.000		0	0	0	FT	0.000	IN
	0.000	0.000		0	0	0	FT	0.000	IN





# PLATE AND FRAME TIGHTNING DIMENSIONS

FORMULAR -  $A = n(3.95 + t) (0.03937)$

A= Dimension in inches. n= number of plates. t= plate thickness. .03937 conv. to in.

Fill in "yellow" area only!						
Plate Stack	Number of plates	MM plate thickness	Plate + Gasket	Tightning Dimension	Tightning Dimension	Fraction Inches
				MM	INCHES	
Flow Plates	38	0.500	4.45	169.1	6.657	TODCO
Return Pit's			3.95	0	0.000	206
<b>TOTAL TIGHTNING DIMENSION</b>				<b>169.1</b>	<b>6.657</b>	<b>6 5/8</b>
Flow Plates	80	0.500	4.45	356	14.016	A-10BFG
Return Pit's	81	0.500	4.45	360.45	14.191	AI/Lav
<b>TOTAL TIGHTNING DIMENSION</b>				<b>716.45</b>	<b>28.207</b>	<b>28 3/16</b>
Flow Plates			3.95	0	0.000	
Return Pit's			3.95	0	0.000	
<b>TOTAL TIGHTNING DIMENSION</b>				<b>0</b>	<b>0.000</b>	<b>0</b>
Flow Plates			3.95	0	0.000	
Return Pit's			3.95	0	0.000	
<b>TOTAL TIGHTNING DIMENSION</b>				<b>0</b>	<b>0.000</b>	<b>0</b>
Flow Plates			3.95	0	0.000	
Return Pit's			3.95	0	0.000	
<b>TOTAL TIGHTNING DIMENSION</b>				<b>0</b>	<b>0.000</b>	<b>0</b>

Weeks  
#302

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 EMAIL: davidradiator@bellsouth.net

Contact: David J. Bienvenu  
 President



ANY  
QUESTIONS?



THANK YOU FOR  
ALLOWING ME TO SHARE



THE END